

POWER PLANT
ENGINEERING REPORT NO. **3A**
(Revised)

STANDARD FIRE TEST
APPARATUS AND PROCEDURE
(For Flexible Hose Assemblies)

Prepared by:
Propulsion Branch
Engineering & Manufacturing Division

Revision **3A** Prepared by:
E. P. Burke

Revision **3A** Approved by:
T. G. Horeff

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Table of Contents

| | <u>Page</u> |
|---|-------------|
| Standard Fire Test Apparatus and Procedure | 1 |
| Appendix I, Description of Test Apparatus | 5 |
| Appendix II, Test Burner Standardization Apparatus | 6 |
| Appendix III, Criteria/Qualified Burners | 15 |
| Table 1. Burner Specifications and Performance | 25 |

List of Figures

| | |
|--|----|
| Figure 1. Hose Assembly Test Bench | 3 |
| Figure 2. Hose Test Control Panel | 4 |
| Figure 3. Burner Standardization Apparatus | 8 |
| Figure 4. Lennox OB-32 Temperature Profile | 19 |
| Figure 5. Carlin 200 CRD Conversion Oil Burner | 20 |
| Figure 6. Carlin 200 CRD Burner | 21 |
| Figure 7. Air Tube Reduction Cone | 22 |
| Figure 8. Burner Tube Extension | 23 |
| Figure 9. Carlin 200 CRD Temperature Profile | 24 |
| Figure 10. Stewart-Warner HPR-250 Conversion Oil Burner | 26 |
| Figure 11. Stewart-Warner HPR-250 Burner | 27 |
| Figure 12. Stewart-Warner HPR-250 Temperature Profile | 28 |
| Figure 13. Stewart-Warner FR-600 Conversion Oil Burner | 29 |
| Figure 14. Stewart-Warner FR-6000 Burner | 30 |
| Figure 15. Stewart-Warner FR-600 Temperature Profile | 31 |

List of Drawings

| | |
|--|----|
| Drawing No. TD271-1A-B. Assembly-BTU Transfer Device for Torch Standardization | 9 |
| Drawing No. TD271-1-A. Reducer-BTU Transfer Device for Torch Standardization | 10 |
| Drawing No. TD271-1-1-B. Inlet Tube-BTU Transfer Device for Torch Standardization | 11 |
| Drawing No. TD271-2-B. Outlet Tube-BTU Transfer Device for Torch Standardization | 12 |
| Drawing No. TD271-3-B. Thermometer Mounting-BTU Transfer Device for Torch Standardization | 13 |
| Drawing No. TD271-4-B. Test Specimen-BTU Transfer Device for Torch Standardization | 14 |

List of Tables

| | |
|---|----|
| Table 1. Burner Specifications and Performance | 25 |
| List of References | 32 |

Table of Contents

| | <u>Page</u> |
|---|-------------|
| Standard Fire Test Apparatus and Procedure | 1 |
| Appendix I, Description of Test Apparatus | 5 |
| Appendix II, Test Burner Standardization Apparatus | 6 |
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List of Figures

| | |
|--|----|
| Figure 1. Hose Assembly Test Bench | 3 |
| Figure 2. Hose Test Control Panel | 4 |
| Figure 3. Burner Standardization Apparatus | 8 |
| Figure 4. Lennox OB-32 Temperature Profile | 19 |
| Figure 5. Carlin 200 CRD Conversion Oil Burner | 20 |
| Figure 6. Carlin 200 CRD Burner | 21 |
| Figure 7. Air Tube Reduction Cone | 22 |
| Figure 8. Burner Tube Extension | 23 |
| Figure 9. Carlin 200 CRD Temperature Profile | 24 |
| Figure 10. Stewart-Warner HPR-250 Conversion Oil Burner | 26 |
| Figure 11. Stewart-Warner HPR-250 Burner | 27 |
| Figure 12. Stewart-Warner HPR-250 Temperature Profile | 28 |
| Figure 13. Stewart-Warner FR-600 Conversion Oil Burner | 29 |
| Figure 14. Stewart-Warner FR-6000 Burner | 30 |
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| | |
|---|----|
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STANDARD FIRE TEST APPARATUS AND PROCEDURE

(For Flexible Hose Assemblies)

Standard Fire Test Apparatus and Procedure

This method of test is intended to determine the fire resistance of flexible hose assemblies under simulated conditions. The test is aimed at producing a typical aircraft powerplant fire, vibration of the type encountered during rough engine operation, and the various flight conditions of fluid flow, pressure, and temperature.

The fire test apparatus described in Appendix I of this report, or equivalent equipment, shall be used for determining fire resistance. The main components of this apparatus are:

- a. Burner: See Appendix III
- b. Vibrating mechanism.
- c. Bench.
- d. Hood.
- e. Temperature measurement and recording.
- f. Oil circulator and heater.

The flame produced by the burner shall be calibrated by means of the standardization device and method described in Appendix II and shall have a minimum **B. t. u.** value of **4500 B. t. u.** per hour.

The length of hose assembly to be tested shall be not less than **24-inches**. The hose shall be mounted horizontally on the test bench (see figure 1) and shall include one full **90°** bend. The hose assembly shall be located inside of the hood except when limited by physical characteristics such as recommended minimum bend radius. The nearest surface of the hose is to be located 4 inches beyond the burner barrel extension. **SAE 20** oil at a temperature of not less than **200°F.** shall be circulated through the hose assembly and system to remove all air from the piping system and to establish the required operating temperature. The oil flow rate and pressure parameters specified in the applicable Technical Standard Order **(TSO)** shall be established and maintained during the test. The vibrating ~~mechanism~~ shall then be started and the assembly checked to make certain that no resonant whipping occurs. The fan which affects the air movement over the assembly shall be started. The actual fire test is then begun by igniting the burner and starting the chronometer simultaneously. A satisfactory and convenient means of accomplishing this is by use of the control panel shown in figure 2.

A hose assembly is considered acceptable if it complies with the test conditions and parameters for the time period specified in the **TSO** without any evidence of leakage. Leakage shall be detected by visual observation from a distance of not more than five feet, at a position where the specimen, flame, and drip pan are visible.

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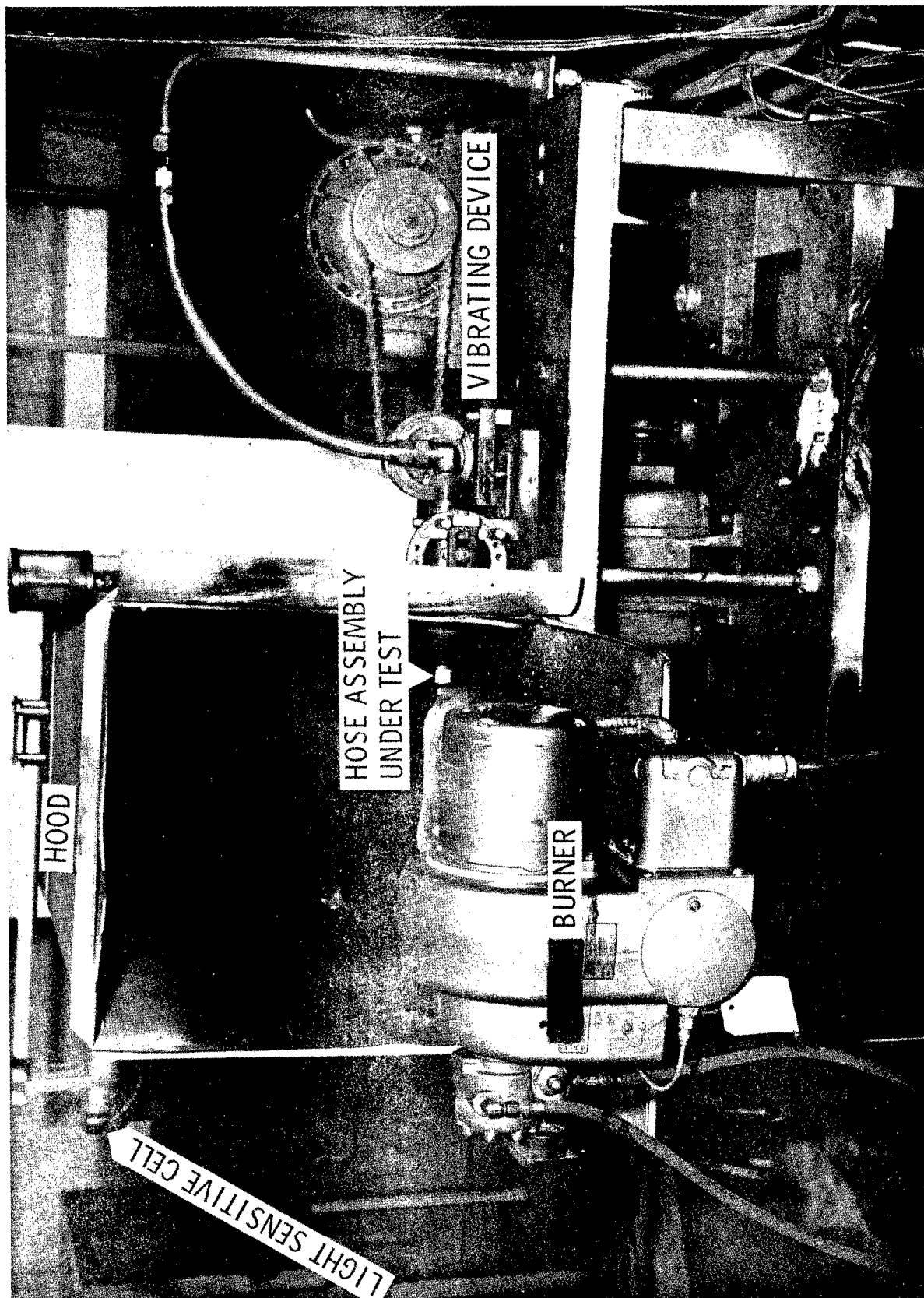


FIGURE 1. HOSE ASSEMBLY TEST BENCH

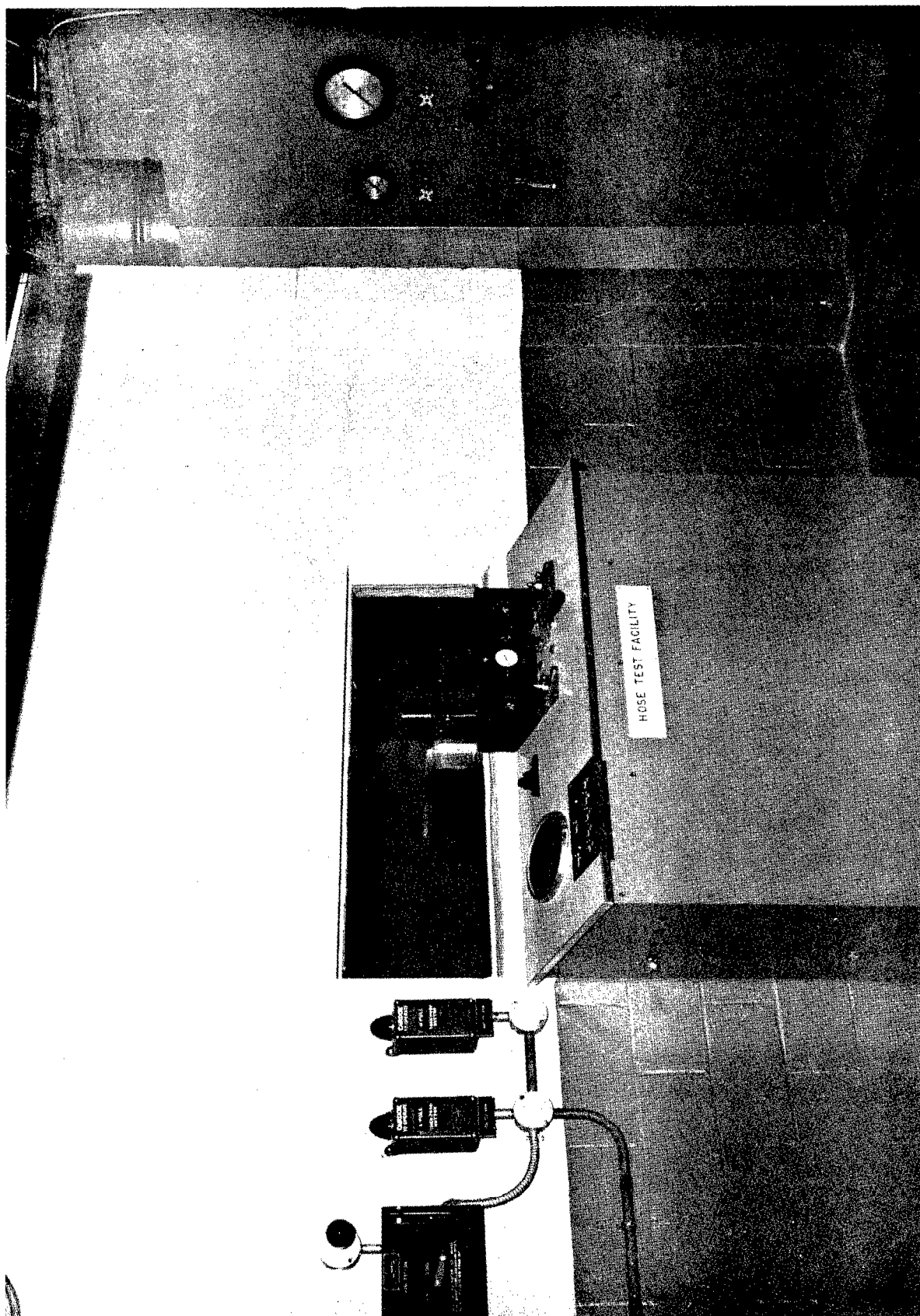


FIGURE 2. HOSE TEST CONTROL PANEL

APPENDIX I

~~Description~~ of Test Apparatus

TEST BURNER - The burner used to produce the flame shall be a modified gun-type conversion oil burner as described in Appendix III or equivalent. With the oil delivery rate at 2 gallons per hour, adjust the inlet air to achieve the controlling characteristics of the flame (see Appendix III). The portion of the flame in which the hose assembly is to be tested should deliver a minimum heat output of **4500 B. t. u.** per hour. Because slight variations in the air pressure as determined by the manometer readings will produce a marked variation in the flame **B. t. u.** output, it may be necessary to vary the air pressure slightly to obtain the required rate of heat transfer.

VIBRATING MECHANISM - The vibrating mechanism is shown in Figure 1. One end of the hose assembly shall be subjected to a total lateral or longitudinal displacement of not less than **1/8** inch (**1/16** inch on each side of normal) at **2000 c.p.m.** The vibrated end of the hose assembly shall be subjected to the flame. The vibrating fixture shall be as light as possible to avoid excessive heat transfer or loss through the fixture.

BENCH - The bench consists essentially of a steel table, **60** inches wide, **28** inches deep, and **32** inches high. Mounted on this bench is the vibrating mechanism and a hood.

HOOD - The hood (see figure 1) is **25** inches wide and **25** inches high. The vibrated fitting is located 7 inches back of the open front of the hood. The rear end of the hood is **ducted** to a fan, which draws air through the hood opening at a velocity of **400 f. p.m.** as measured by an **Alnor velometer** located at the hose assembly specimen. This air movement aids in keeping the flame horizontal and in exhausting fumes.

TEMPERATURE MEASUREMENT AND RECORDING - The temperature sensing system including the thermocouples and indicator shall have an allowable overall error of ± 1 (one) percent at **2000°F.** The flame temperature shall be measured **four** inches from the end of the burner barrel extension. A sufficient number of the thermocouples shall be used to assure that the specified temperature exists at least along the entire end fitting and the hose for a distance of not less than five inches.

OIL CIRCULATOR AND HEATER - The oil circulating and heating **equipment** consists of an electrically driven oil pump and an oil tank with a thermostatically controlled immersion heater. The plumbing of the oil system also includes pressure relief valves, flow indicators, pressure **gages**, and control and selector valves.

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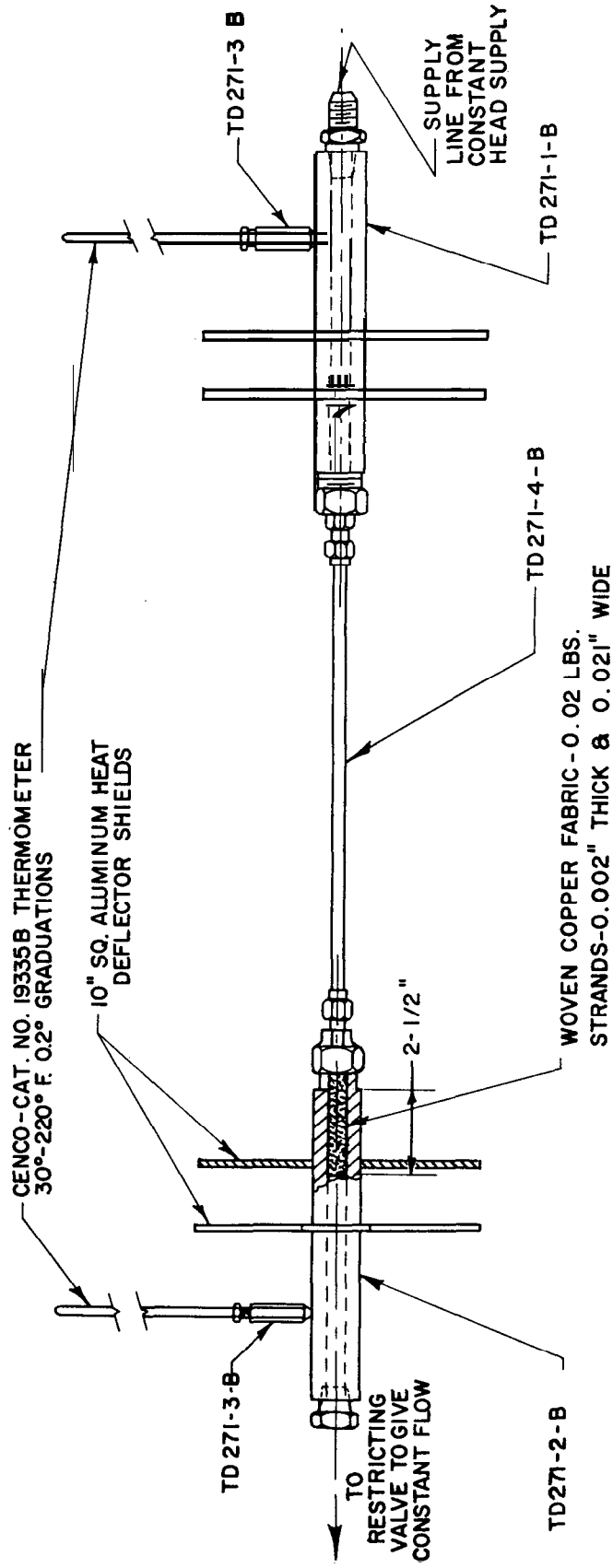
OIL CIRCULATOR AND HEATER - The oil circulating and heating **equipment** consists of an electrically driven oil pump and an oil tank with a thermostatically controlled immersion heater. The plumbing of the oil system also includes pressure relief valves, flow indicators, pressure **gages**, and control and selector valves.

After the warmup period, the temperatures indicated by the inlet and outlet thermometers are recorded every ~~1/2-minute~~ during a ~~3-minute~~ period. The average difference in temperature ($^{\circ}\text{F.}$) of the inlet and outlet water multiplied by the rate of the water flow (~~500~~ pounds per hour) equals the rate of **B. t. u.** increase of the water flowing through the device, and this value is an ~~indication~~ of the severity of the portion of the flame in which hose assemblies are tested.

In addition to these general procedures, those specific procedures called for in ~~TSO-C42~~, Propeller Feathering Hose Assemblies and ~~TSO-C53a~~, Fuel Engine Oil System Hose Assemblies shall be followed as applicable.

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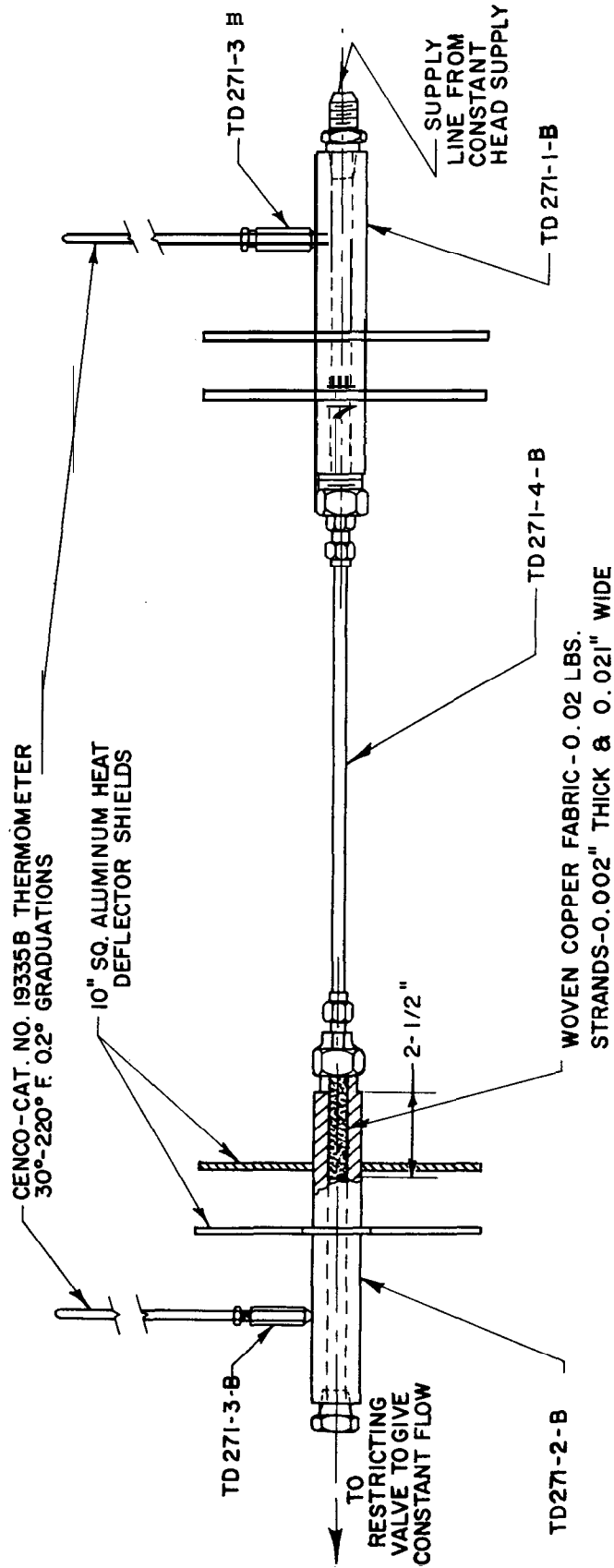


ASSEMBLY

B.T.U. TRANSFER DEVICE FOR TORCH STANDARDIZATION

Civil Aeronautics Administration
 Technical Development and Evaluation Center
 Indianapolis, Ind.

Drawing No. TD271-1A-B

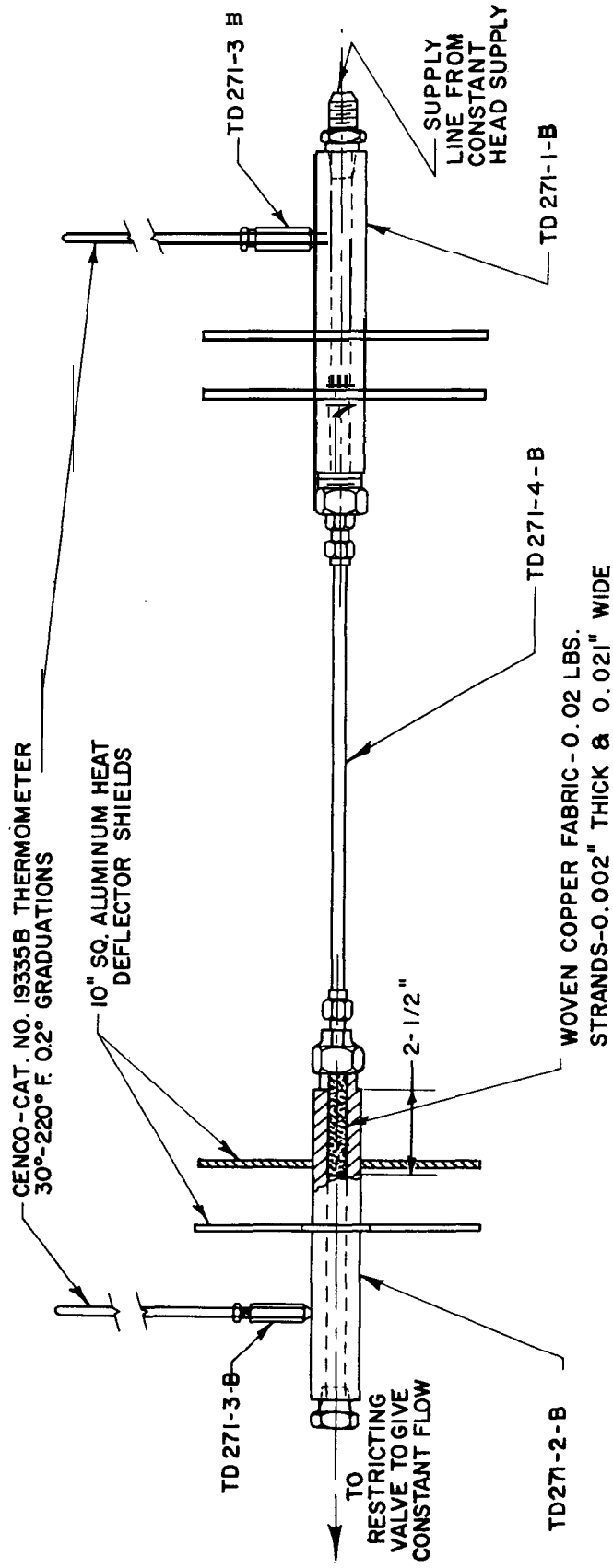


ASSEMBLY

B.T.U. TRANSFER DEVICE FOR TORCH STANDARDIZATION

Civil Aeronautics Administration
Technical Development and Evaluation Center
Indianapolis, Ind.

Drawing No. TD271-1A-B



ASSEMBLY

B.T.U. TRANSFER DEVICE FOR TORCH STANDARDIZATION

Civil Aeronautics Administration
 Technical Development and Evaluation Center
 Indianapolis, Ind.

Drawing No. TD271-1A-B

9"

1-1/4"

53°

1/8"

1.062 DIA

12"

3/4" MALE / 1/2" FEMALE BUSHING

TAP 1/8 - 27 NPT

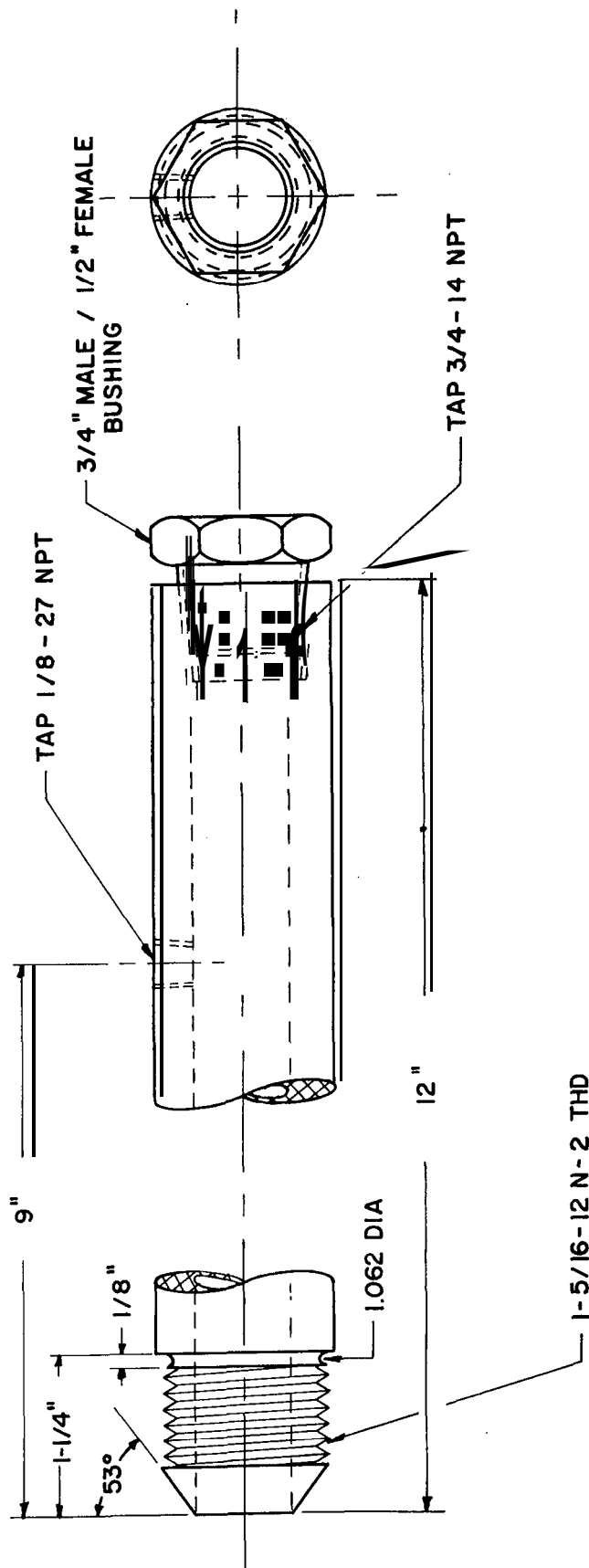
TAP 3/4 - 14 NPT

1-5/16 - 12 N - 2 THD

**B.T.U. TRANSFER DEVICE FOR
TORCH STANDARDIZATION**

Drawing No. TD271-2-B

MATL. — DILECTO ASBESTOS BASE TUBING, AA-79
 Made by Continental-Diamond Fiber Corp., Newark, Del.
 1 7/16" O.D. x 13/16" I.D. x 12"

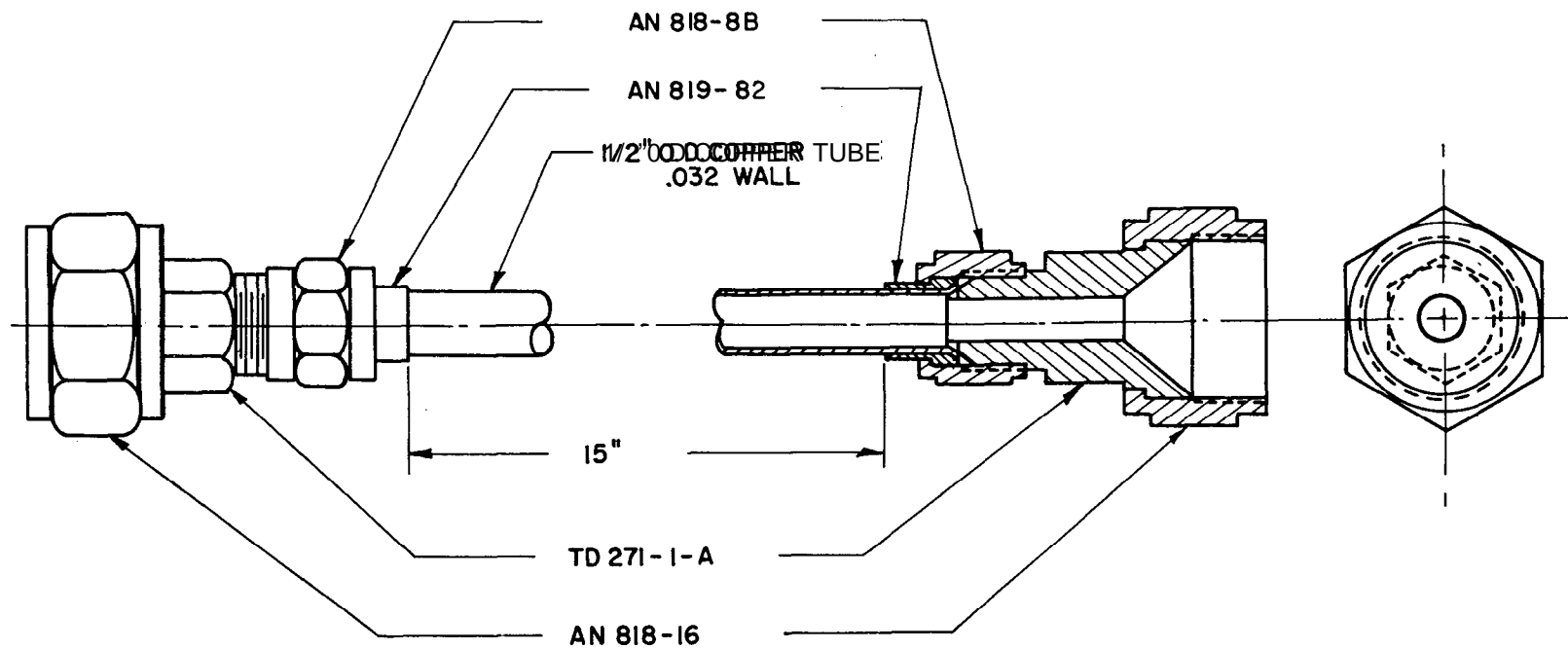


OUTLET TUBE

B.T.U. TRANSFER DEVICE FOR TORCH STANDARDIZATION

Civil Aeronautics Administration
 Technical Development and Evaluation Center
 Indianapolis, Ind.

Drawing No. TD271-2-8

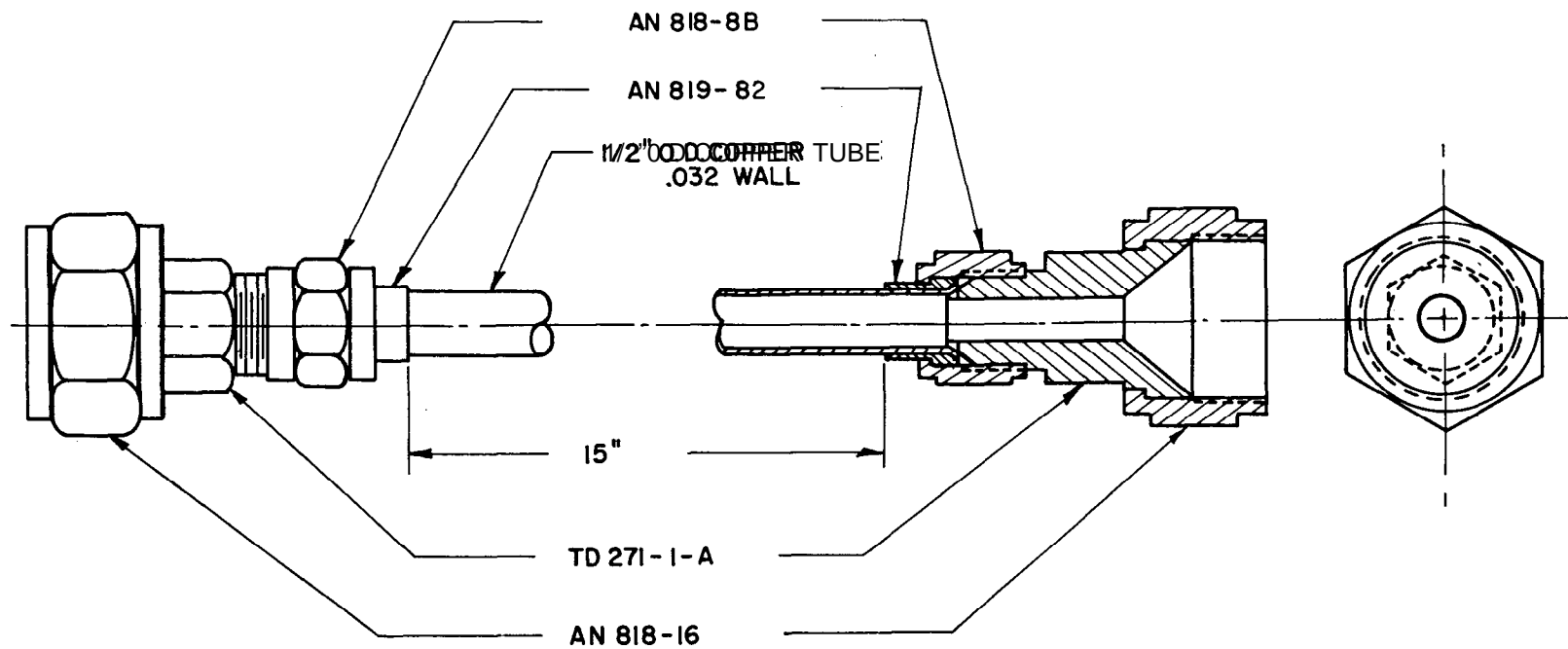


TEST SPECIMEN

**B.T.U. TRANSFER DEVICE FOR
TORCH STANDARDIZATION**

Civil Aeronautics Administration
Technical Development and Evaluation Center
Indianapolis, Ind.

Drawing No. TD271-1-4-B



TEST SPECIMEN

**B.T.U. TRANSFER DEVICE FOR
TORCH STANDARDIZATION**

Civil Aeronautics Administration
Technical Development and Evaluation Center
Indianapolis, Ind.

Drawing No. TD271-1-4-B

4. The air tube diameter was decreased to 2 1/2 inches starting 1 1/2 inches forward of the fuel nozzle tip. The reducing cone is shown in figure 7.

5. Two ~~1/16-inch-thick~~ by ~~3/4-inch-wide~~ stainless steel fuel deflectors were installed at the 3 and 9 o'clock positions with their ends ~~5/8-inch~~ from the fuel nozzle centerline and 3 ~~/4-inch~~ forward of the fuel nozzle tip. A 1-inch wide stainless steel fuel deflector (~~1/16-inch~~ thick) was installed at the 12 o'clock position with its edge 3/4 inch forward of the fuel nozzle tip and ~~3/4-inch~~ above fuel nozzle centerline.

6. A static pressure port was installed 1-inch forward of the air tube mounting flange.

7. A ~~12 1/2-inch~~ burner extension (reference 1) shown in figure 8 was added to the end of the burner air tube. The extension was installed on the air tube so that the wide end was 10 inches beyond the end of the air tube. The air pressure was adjusted to indicate 0.37 inch H₂O air tube pressure.

The temperature profile obtained with this burner configuration is shown in figure 9. A heat transfer rate of 4, 545 B. t. u. /hr. to the ~~1/2-inch~~ tube was obtained as shown in table 1, and the total thermal energy measured an average range from 9.3 to 11.2 B. t. u. /ft /sec. Oxygen volumetric concentration within the flame ranged from 6.5 percent to 8.5 percent.

STEWART-WARNER MODEL HPR 250, manufactured by the Stewart-Warner Corporation, Heating and Air Conditioning Division, Lebanon, Indiana 46052, figures 10 and 11, was modified in the following manner to produce a diffused ~~6-inch~~ (vertical) by ~~11-inch~~ (horizontal) size flame with homogeneous temperature gradient:

1. An 80 fuel nozzle rated at 2.25 gal/hr. and pressure adjusted to 95 psig and delivering 2.04 gal/hr. was installed.

2. The air cone assembly was removed.

3. The air tube diameter was reduced to 2 1/2 inches starting 1 1/2 inches forward of the fuel nozzle tip with the addition of the reducing cone shown in figure 7.

4. Four ~~1/16-inch~~ by ~~3/4-inch~~ stainless steel fuel deflector were mounted on the reducing cone at ~~3, 6, 9~~ and ~~12~~ o'clock positions. The deflector edges were ~~3/4-inch~~ from the fuel nozzle centerline and ~~3/4-inch~~ forward of the fuel nozzle tip.
5. A static air-pressure port was installed 1-inch forward of the burner tube mounting flange.
6. A ~~12 1/2-inch~~ burner extension (reference ~~1~~) shown in figure 8 was added to the end of the burner air tube. The extension was installed so that the wide end was ~~10~~ inches from beyond the end of the air tube. Air pressure in the tube was adjusted to ~~0.14~~ inch H₂O.

The temperature profile shown in figure ~~12~~ was obtained with this configuration, and a heat transfer rate of ~~4,646~~ B. t. u. /hr. to the ~~1/2-inch~~ tube was obtained as shown in table ~~1~~. The total thermal energy measured an average range from ~~9.3~~ to ~~10.1~~ B. t. u. /ft²/sec.. The O₂ concentration, measured through the horizontal centerline of the flame, fluctuated from ~~9.2~~ percent to ~~9.5~~ percent in the flame center. Two inches from each side, the O₂ concentration increased to ~~11~~ percent and ~~15~~ percent, respectively. This burner had the tendency to burn richer at the outer edges of the flame, as observed visually and confirmed by O₂ concentration measurements.

STEWART-WARNER MODEL FR-600, as shown in figures ~~13~~ and ~~14~~, was ~~modified in~~ the following manner to produce a diffused ~~6-inch~~ (vertical) by ~~11-inch~~ (horizontal) size flame with homogeneous temperature gradient.

1. An ~~80°~~ fuel nozzle rated at ~~2.25~~ gal/hr. and pressure adjusted to deliver ~~2.03~~ gal/hr. at ~~100~~ psig was installed.
2. The flame retention hood assembly was removed.
3. The air tube diameter was decreased to 2 ~~1/2~~ inches starting 1 ~~1/2~~ inches forward of fuel nozzle tip, with the reducing cone as shown in figure ~~7~~.
4. Four ~~1/16-inch~~ by ~~3/4-inch~~ stainless steel deflector were mounted on the reducing cone at ~~3, 6, 9, and 12~~ o'clock positions. The deflector edges were adjusted to within ~~3/4-inch~~ from the fuel nozzle centerline and 1 ~~1/2-inch~~ forward of the fuel nozzle.

4. Four ~~1/16-inch~~ by ~~3/4-inch~~ stainless steel fuel deflectors were mounted on the reducing cone at ~~3, 6, 9~~ and ~~12~~ o'clock positions. The deflector edges were ~~3/4-inch~~ from the fuel nozzle centerline and ~~3/4-inch~~ forward of the fuel nozzle tip.
5. A static air-pressure port was installed 1-inch forward of the burner tube mounting flange.
6. A ~~12 1/2-inch~~ burner extension (reference ~~1~~) shown in figure 8 was added to the end of the burner air tube. The extension was installed so that the wide end was 10 inches from beyond the end of the air tube. Air pressure in the tube was adjusted to ~~0.14~~ inch H₂O.

The temperature profile shown in figure 12 was obtained with this configuration, and a heat transfer rate of ~~4,646~~ B. t. u. /hr. to the ~~1/2-inch~~ tube was obtained as shown in table 1. The total thermal energy measured an average range from ~~9.3~~ to ~~10.1~~ B. t. u. /ft²/sec.. The O₂ concentration, measured through the horizontal centerline of the flame, fluctuated from ~~9.2~~ percent to ~~9.5~~ percent in the flame center. Two inches from each side, the O₂ concentration increased to 11 percent and 15 percent, respectively. This burner had the tendency to burn richer at the outer edges of the flame, as observed visually and confirmed by O₂ concentration measurements.

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1. An 80° fuel nozzle rated at ~~2.25~~ gal/hr. and pressure adjusted to deliver ~~2.03~~ gal/hr. at 100 psig was installed.
2. The flame retention hood assembly was removed.
3. The air tube diameter was decreased to 2 1/2 inches starting 1 1/2 ~~inches~~ forward of fuel nozzle tip, with the reducing cone as shown in figure 7.
4. Four ~~1/16-inch~~ by ~~3/4-inch~~ stainless steel deflectors were mounted on the reducing cone at ~~3, 6, 9, and 12~~ o'clock positions. The deflector edges were adjusted to within ~~3/4-inch~~ from the fuel nozzle centerline and 1 1/2 ~~inches~~ forward of the fuel nozzle.

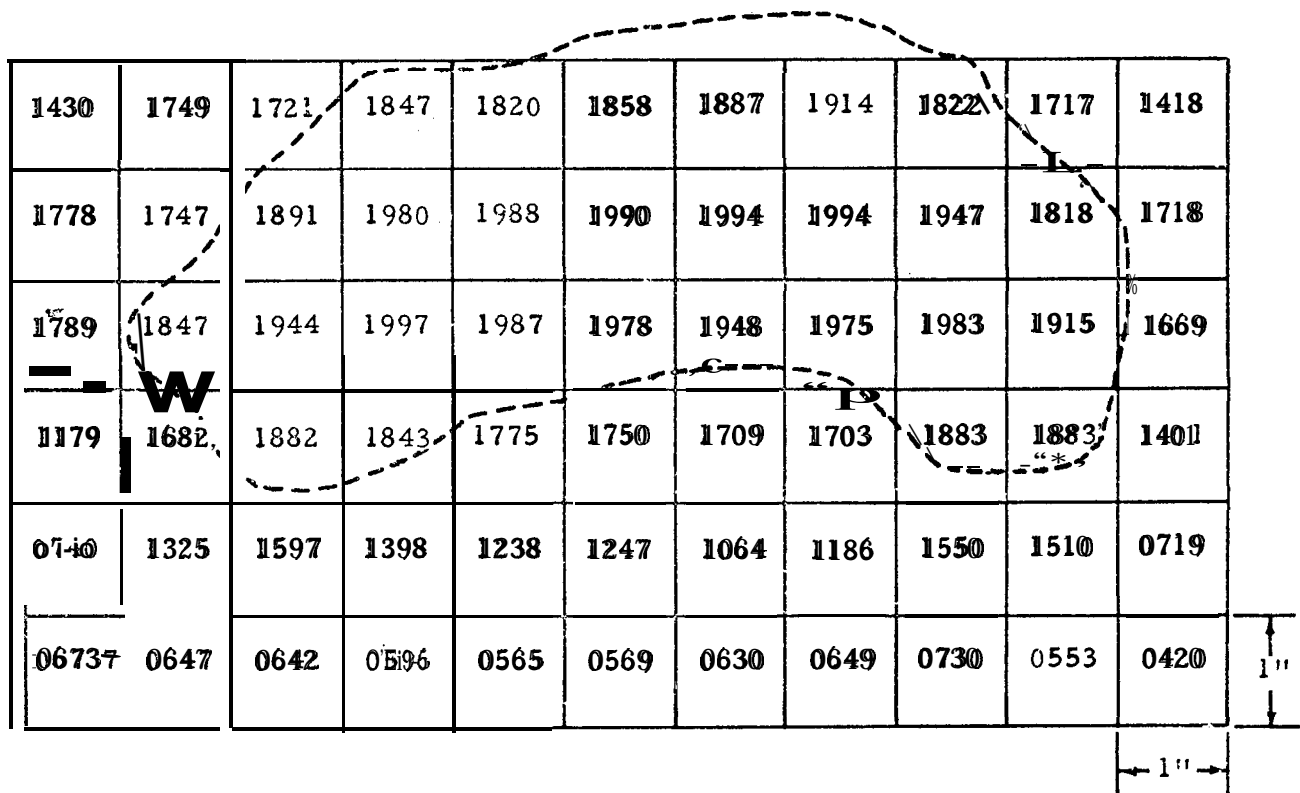
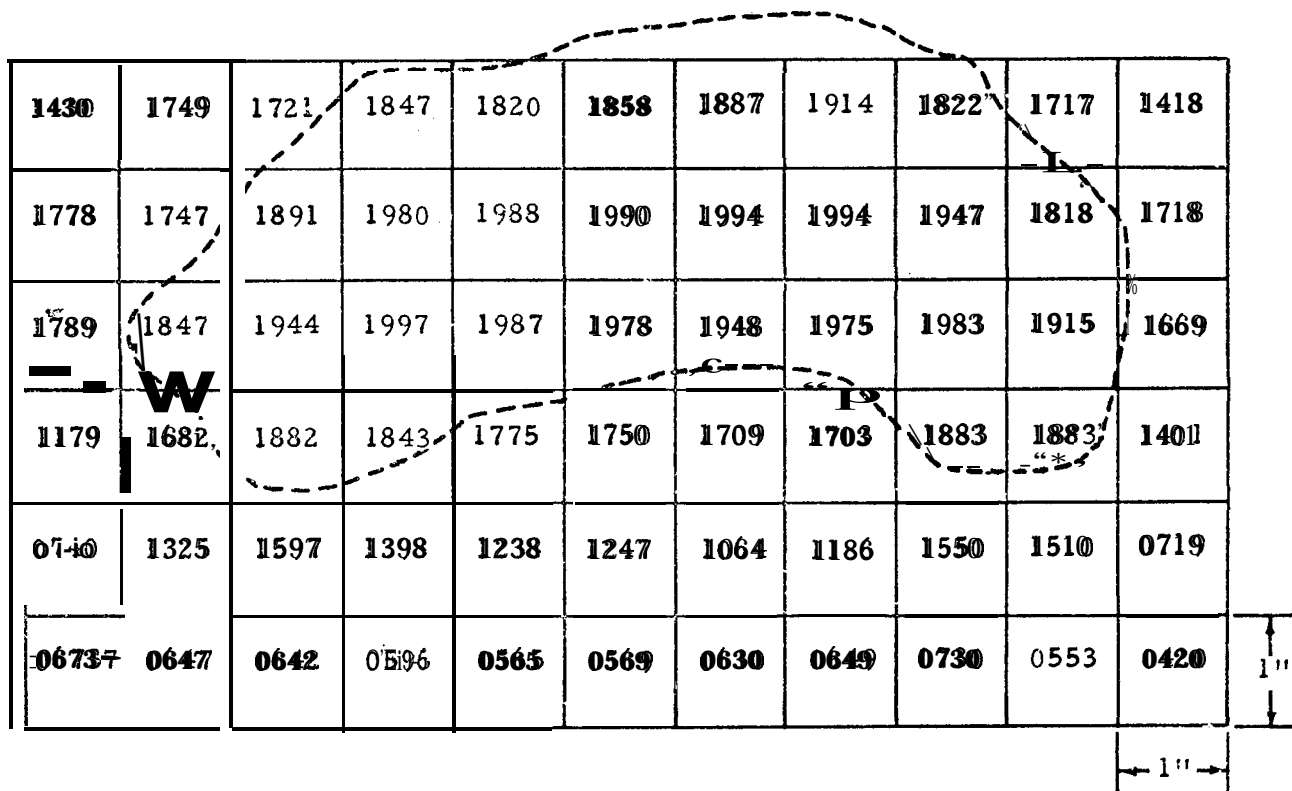


FIGURE 4. LENNOX OB-32 TEMPERATURE PROFILE



----- = 1800°F

FIGURE 4. LENNOX OB-32 TEMPERATURE PROFILE

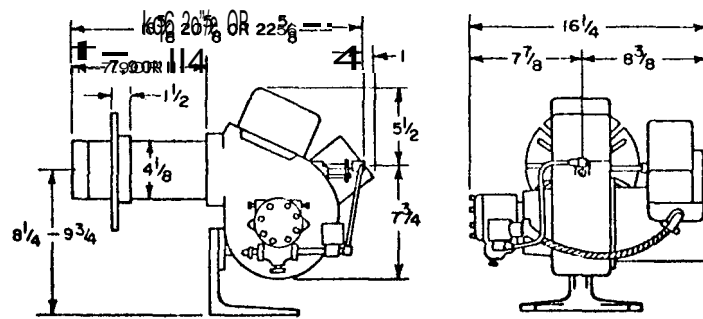


FIGURE 6. CARLIN 200 CRD BURNER

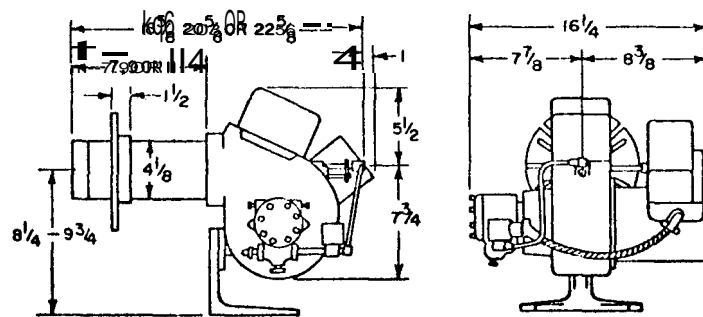


FIGURE 6. CARLIN 200 CRD BURNER

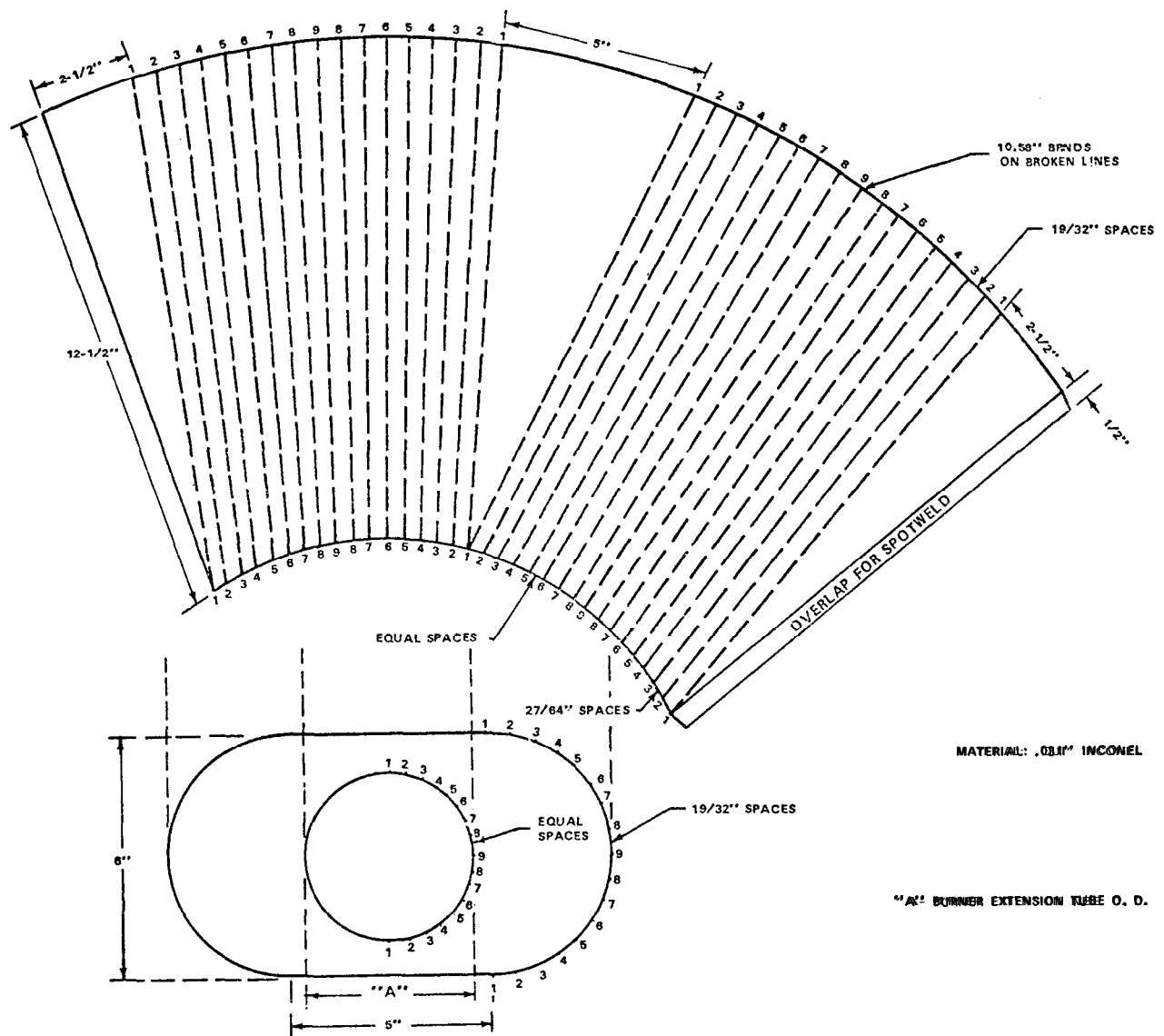


FIGURE 8. BURNER TUBE EXTENSION

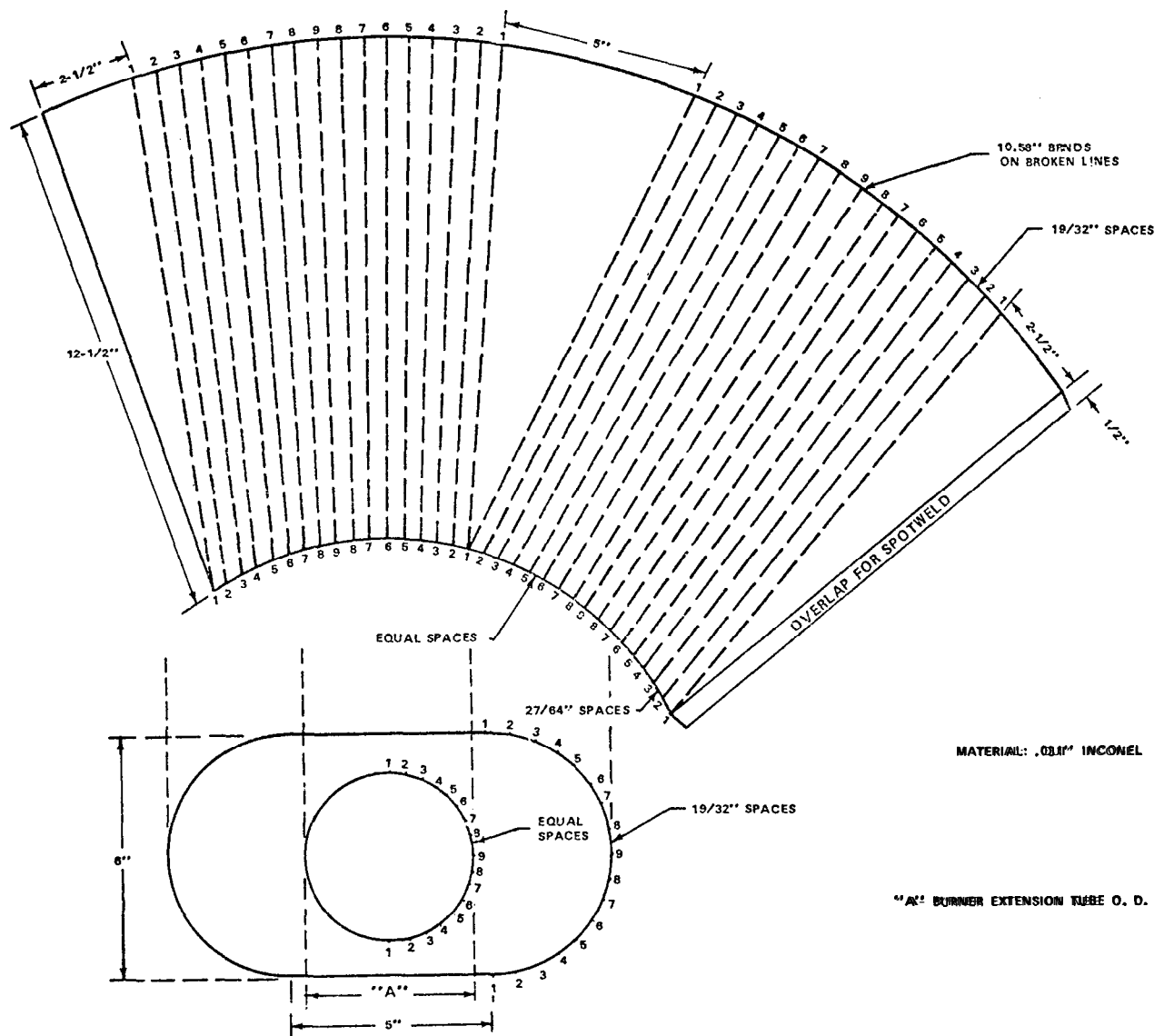


FIGURE 8. BURNER TUBE EXTENSION

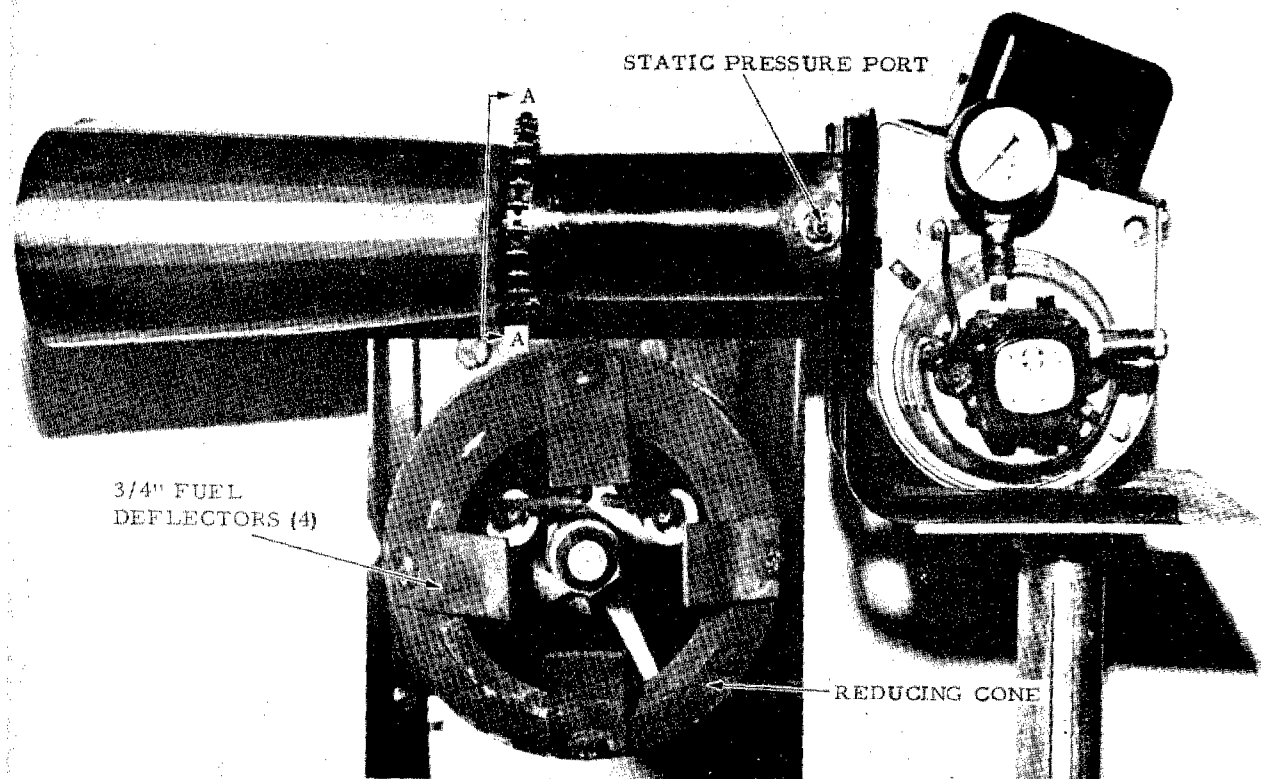


FIGURE 10. STEWART-WARNER ~~HPR-250~~ CONVERSION OIL BURNER

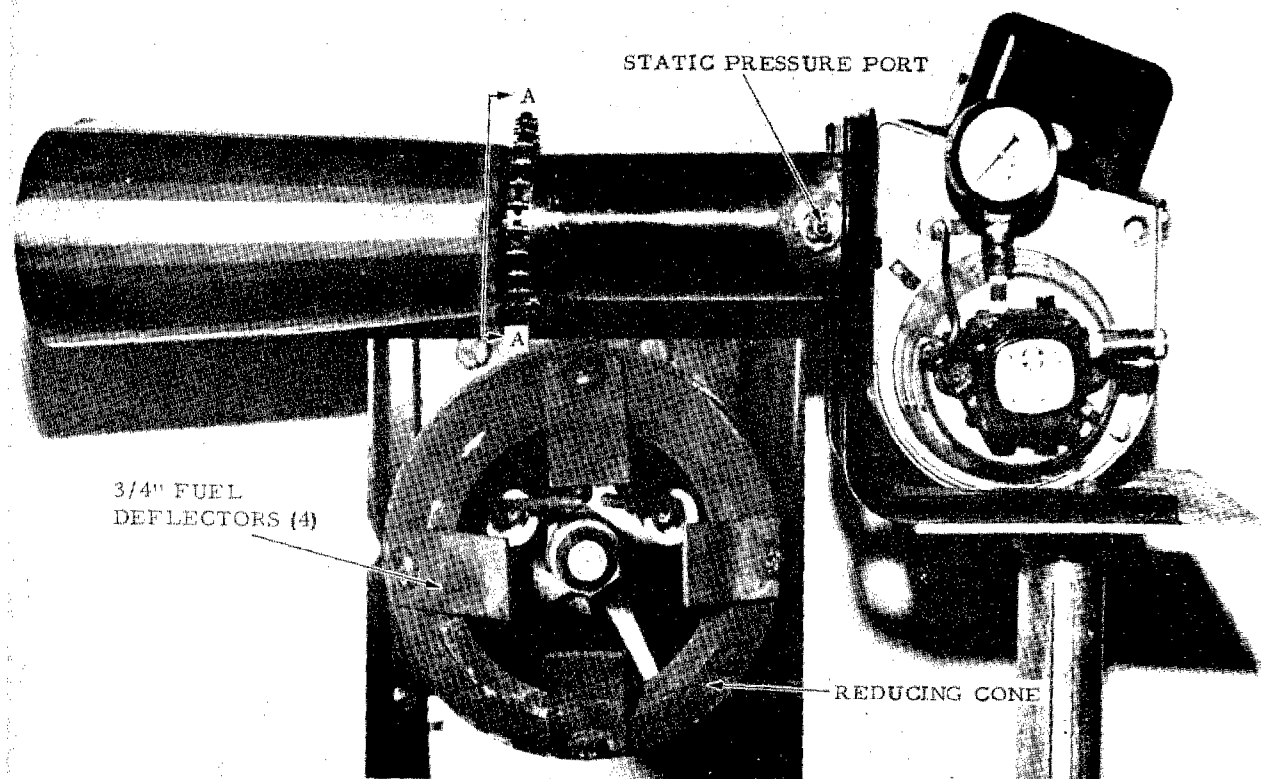


FIGURE 10. STEWART-WARNER ~~HPR-250~~ CONVERSION OIL BURNER

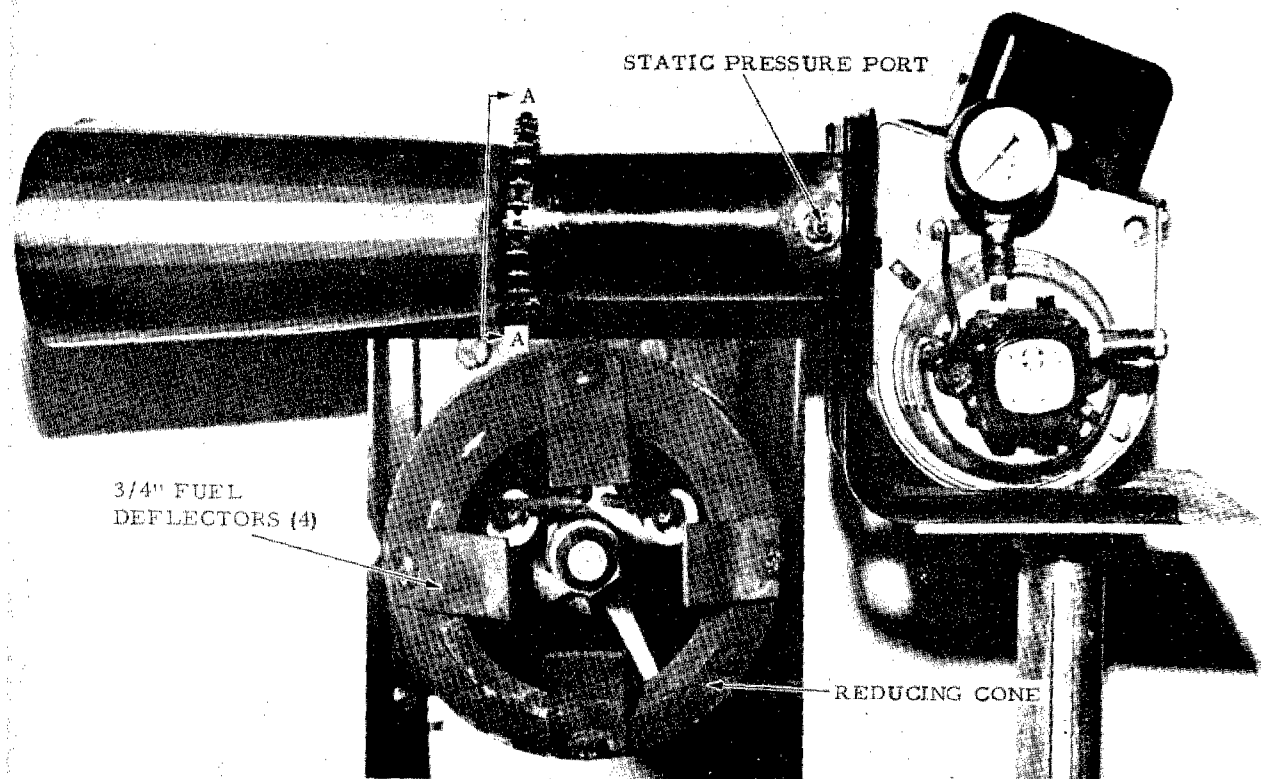


FIGURE 10. STEWART-WARNER ~~HPR-250~~ CONVERSION OIL BURNER

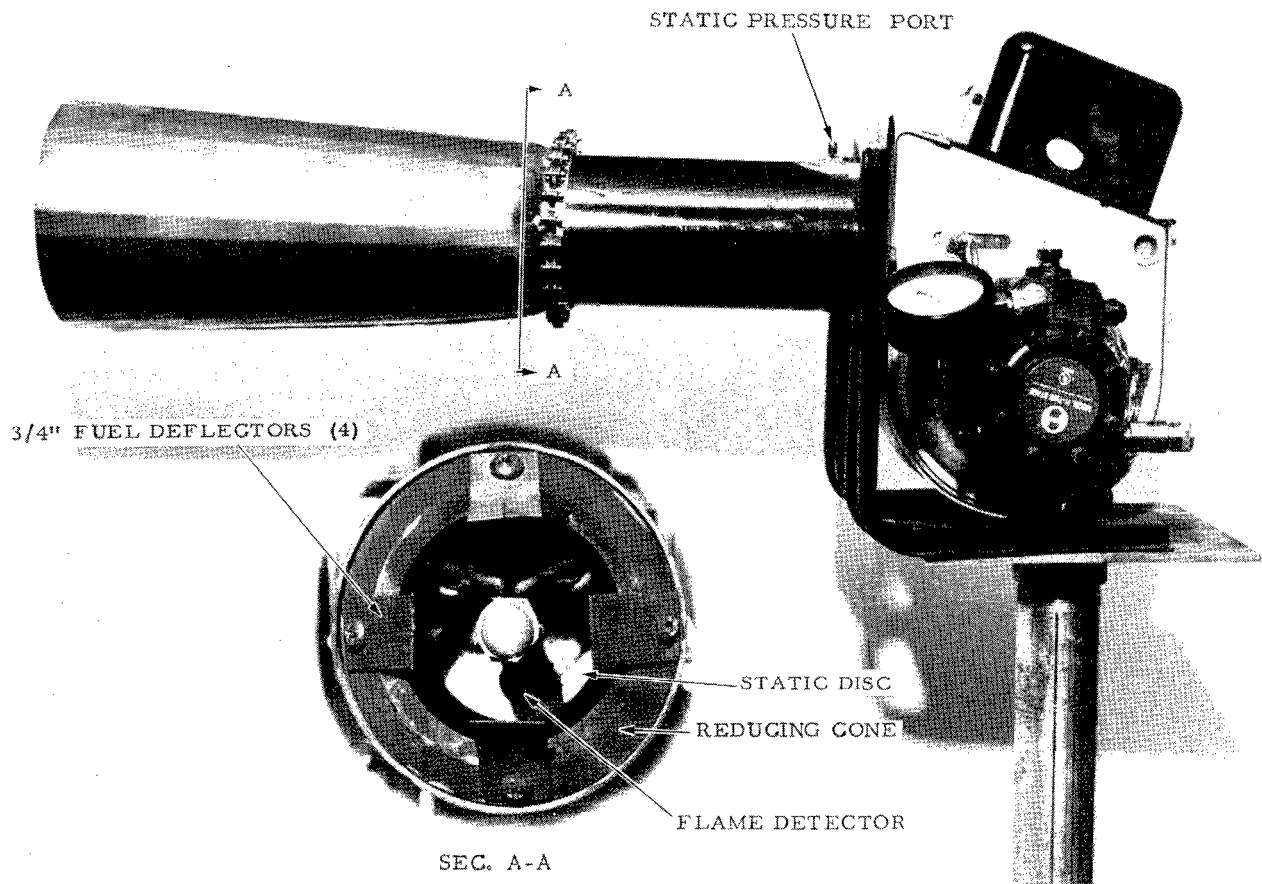
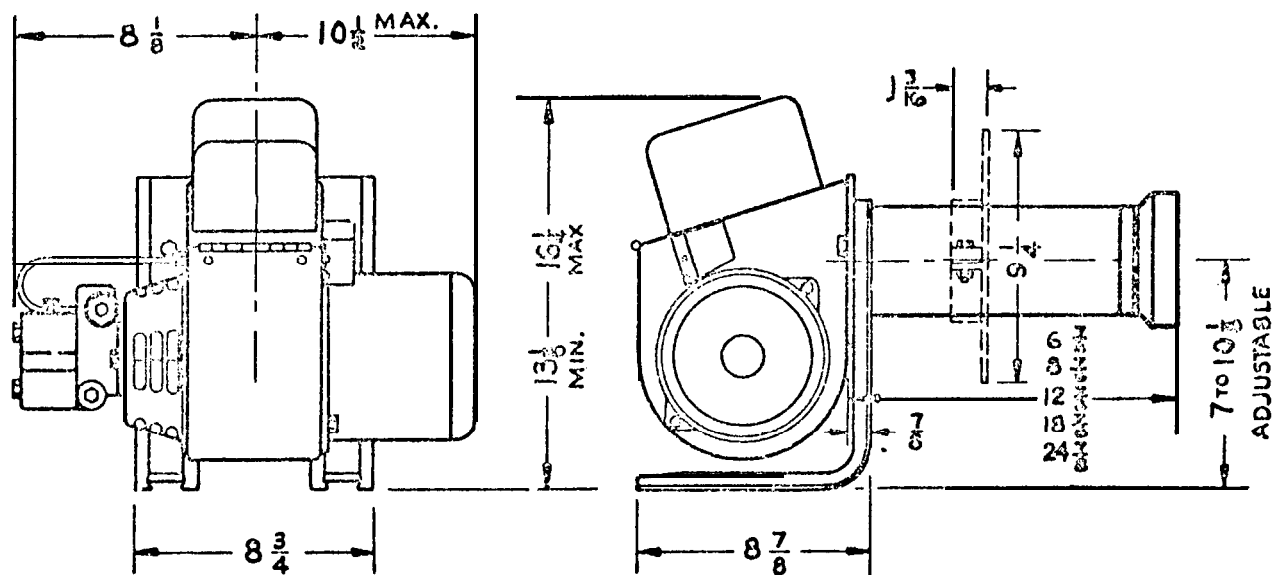


FIGURE 13. STEWART-WARNER FR-600 CONVERSION OIL BURNER



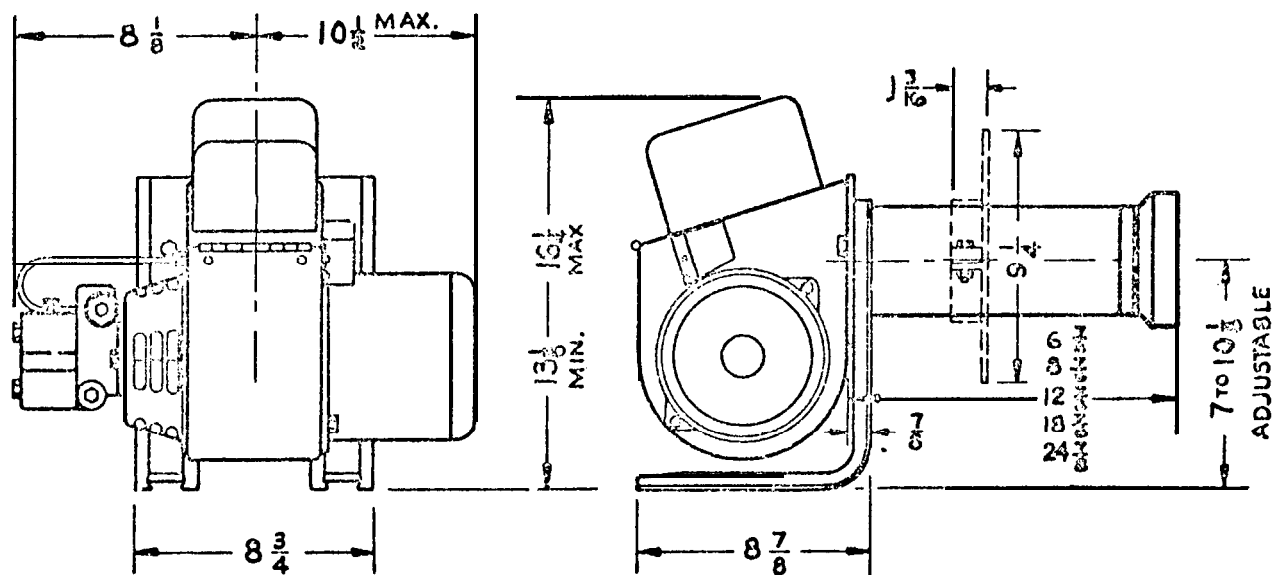


FIGURE 14. STEWART-WARNER FR-600 BURNER

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STANDARD FIRE TEST APPARATUS AND PROCEDURE
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